

Today: **Reference Tracking / Integral Control**

Lecture 17

ECE 147B

Digital Control

Reading: FPW 8.4.1, 8.5.1 <or> FV 9.3, 10.4.3, p.370
 <AND> RS-14, RS-15

Topics: - Reference Tracking
 - Integral Control

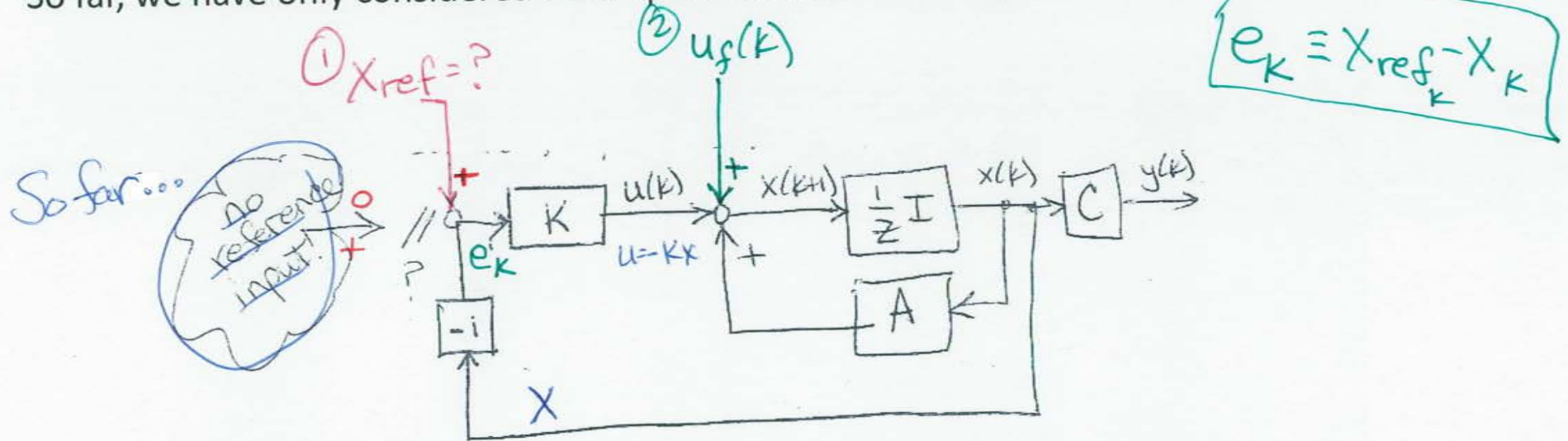
Next Class:

- Optimal Control and Estimation

FPW 9.3.5, 9.4.1 <or> FV 10.2-10.4
<AND> RS-16

Reference Tracking

So far, we have only considered state space control for a "regulator":



Questions: How can we

- 1) Compare $\underline{x(k)}$ (or an ESTIMATE of $x(k)$) with a desired REFERENCE, $\underline{x_r(k)}$?

Given what we have is some $r_y(k)$ to match output $y(k)$.

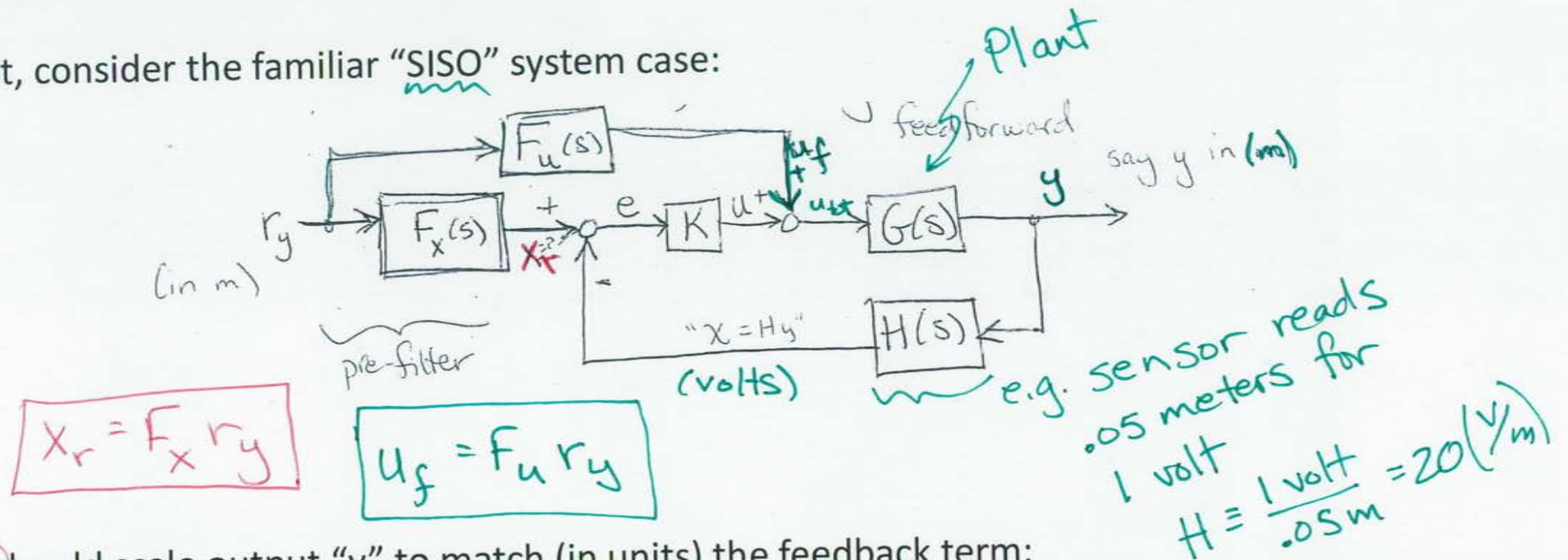
- 2) Generate a feedforward term, $\underline{u_f(k)}$, to eliminate steady state error?

Without feedforward, $u = -K(x - x_{ref})$, means

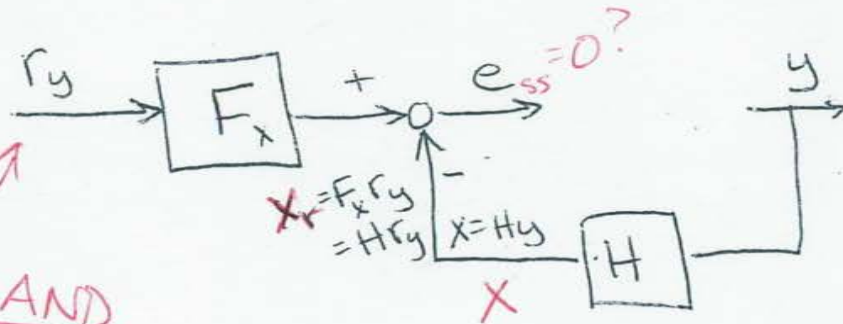
$u = 0$ when $(x - x_{ref}) = 0 \dots$ So if $x = x_{ref}$ requires some force (e.g., pushing a spring), we will have s.s. error.

Reference Tracking

First, consider the familiar "SISO" system case:



F_x should scale output " y " to match (in units) the feedback term:



[Note for Type 1 ($\frac{1}{s}$ higher) $G(s)$, $e_{ss} = 0$ w/out "feedforward"]

(match scaling AND dimensionality...)
(...later when we move to MIMO)

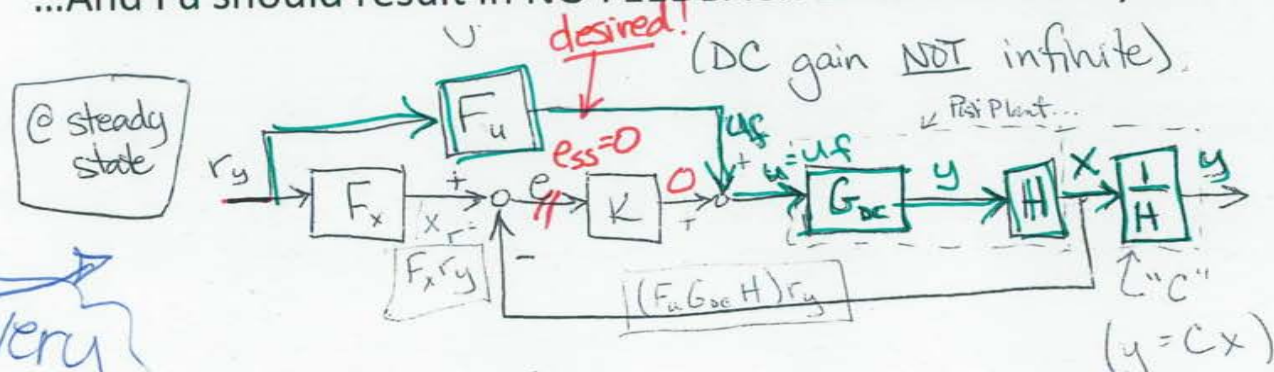
$e_{ss} = 0$, desired
 $0 = F_x r_{y_{ss}} - H y_{ss}$
 $F_x r_{y_{ss}} = H y_{ss} \rightarrow \boxed{F_x = H}$

And we want
 $\boxed{y_{ss} = r_{y_{ss}}}$

Reference Tracking

G_{DC} = DC gain of $G(s)$
(Thinking about "steady state", ss.)
 $\leftarrow F_x = H$, last page...

...And F_u should result in NO FEEDBACK ERROR at steady state:



$$(F_x - F_u G_{dc} H) r_y = 0$$

$$(H - F_u G_{dc} H) r_y = 0$$

$$\therefore F_u = \frac{1}{G_{dc}}$$

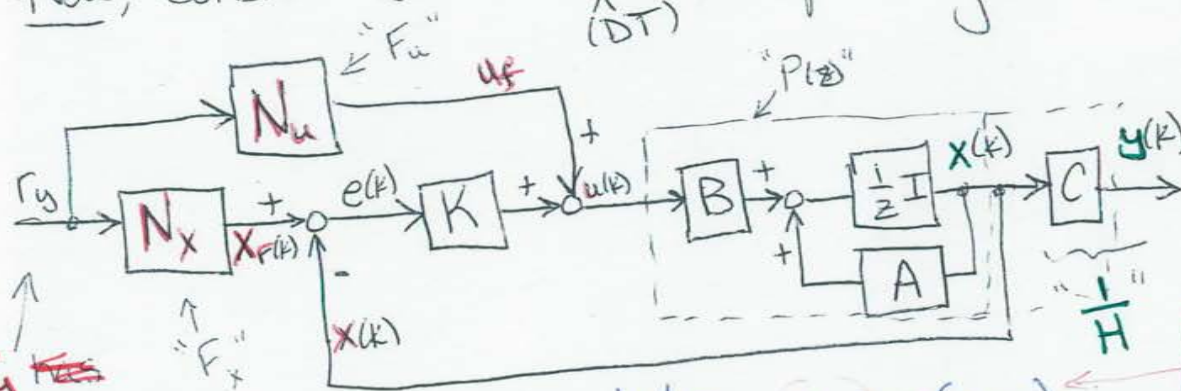
"invert" the plant
(or at least its DC gain).

• w/ $F_u = 0$, $e_{ss} \neq 0$.

• If we want $e_{ss} = 0$, $[F_x r_y - (F_u G_{dc} H) r_y] = 0$

MIMO equivalent:

Now, consider a MIMO state space system form:



* Assume some r_y (to match) y is known (given).

$$x_r = N_x r_y$$

(n x 1) (n x 1) (l x 1)

$$u_f = N_u r_y$$

(m x 1) (m x 1) (l x 1)

r_y (dimensionality & scaling) "matches" y

n states, x is (n x 1)
m inputs, u is (m x 1)
l outputs, y is (l x 1)

to match y

Reference Tracking

(n^{th} -order,
 $\frac{n}{2}$ masses)

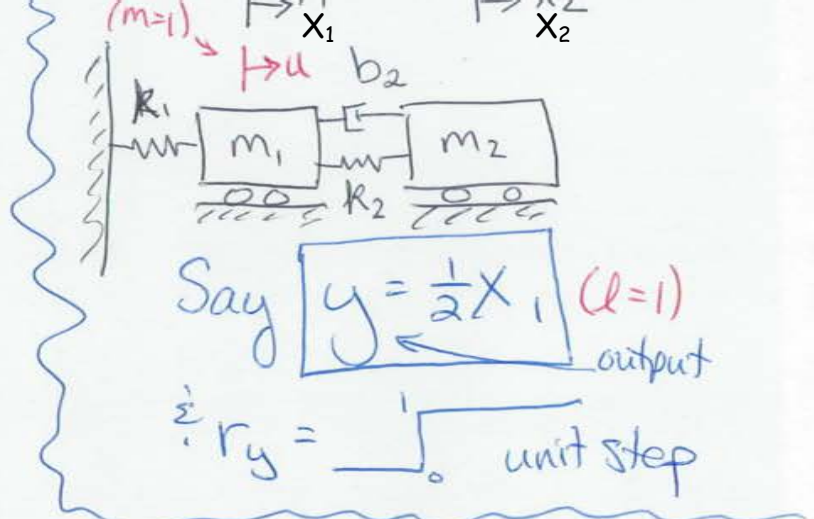
Example: 4th-order spring-mass system

$$\mathbf{X} = \begin{bmatrix} x_1 \\ \dot{x}_1 \\ x_2 \\ \dot{x}_2 \end{bmatrix} \quad (n \times 1), \quad \mathbf{A} = \begin{bmatrix} 0 & 1 & 0 & 0 \\ -\frac{k_1+k_2}{m_1} & -\frac{b_2}{m_1} & \frac{k_2}{m_1} & \frac{b_2}{m_1} \\ 0 & 0 & 0 & 1 \\ \frac{k_2}{m_2} & \frac{b_2}{m_2} & -\frac{k_2}{m_2} & -\frac{b_2}{m_2} \end{bmatrix} \quad (n \times n)$$

$m=1$
input, u

$$\mathbf{B} = \begin{bmatrix} 0 \\ \frac{1}{m_1} \\ 0 \\ 0 \end{bmatrix} \quad (n \times m), \quad \mathbf{C} = \begin{bmatrix} \frac{1}{2} & 0 & 0 & 0 \end{bmatrix} \quad (l \times n)$$

$\uparrow y = \frac{1}{2}x_1$



$$m_1 \ddot{x}_1 + b_2(\dot{x}_1 - \dot{x}_2) + k_2(x_1 - x_2) + k_1 x_1 = u$$

$$\ddot{x}_1 = \left(\frac{1}{m_1}\right) * [\text{stuff}]$$

$P(s)$ [This is a SISO system...]

$$\frac{Y(s)}{U(s)} = \frac{K_{\text{top}}}{s^4 + \dots + k_1}$$

($K_{\text{top}} = \frac{1}{2}$)

$$G_{\text{DC}} = \frac{K_{\text{top}}}{k_1}$$

No "free integrator"! Type 0. Needs feed forward to achieve zero steady-state error.

Recall, $F_u = \frac{1}{G_{\text{DC}}}$; here, $N_u = \frac{1}{G_{\text{DC}}} = \frac{1}{\frac{1}{2k_1}} = 2k_1$

Also, $y_{\text{ss}} = 1 = Cx_{\text{ss}}$ means...

$$(y_{\text{ss}} = \frac{1}{2}x_{1,\text{ss}}), \quad \boxed{x_{1,\text{ss}} = 2} \quad \text{and} \quad \boxed{x_{2,\text{ss}} = x_{1,\text{ss}}}$$

$$\therefore \mathbf{N}x = \begin{bmatrix} 2 \\ 0 \\ 2 \\ 0 \end{bmatrix}$$

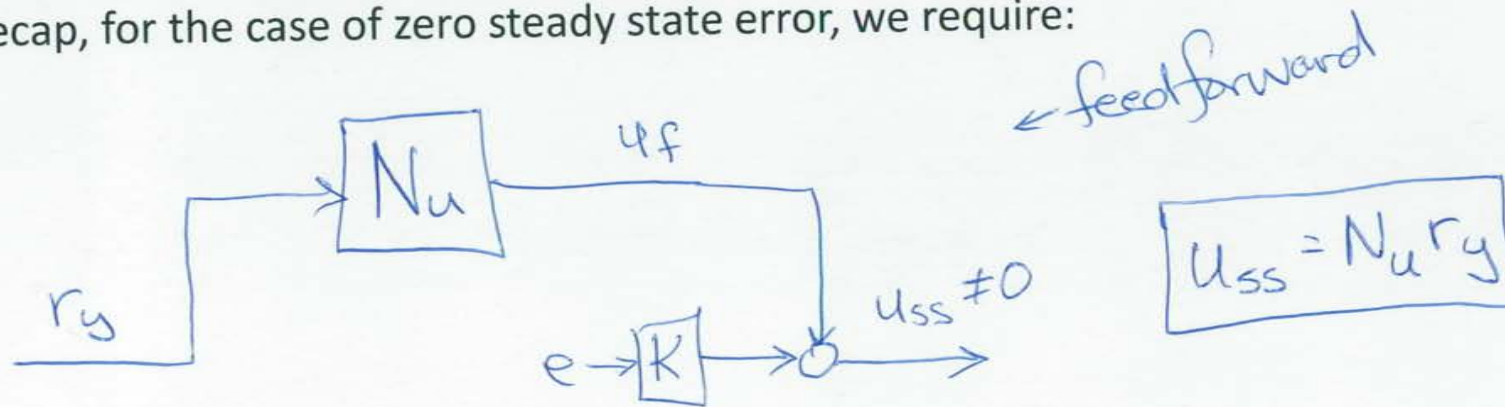
For a spring $F = -k\Delta x$,

So this makes sense. ☺

Both ss positions are 2. Vels are 0.

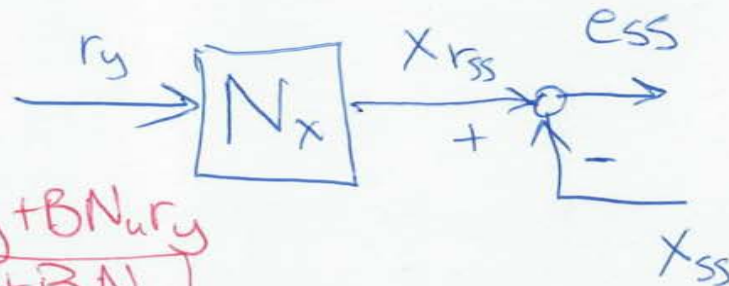
Reference Tracking

Recap, for the case of zero steady state error, we require:



Matching scaling & dimensionality for x_{ss} (vs. y_{ss})

requires a filter on r_y :
($\hat{n}$ pre-filter)



@ ss

$$0 = (A - I)N_x r_y + B N_u r_y$$

$$0 = (A - I)N_x + B N_u$$

$$y_{ss} = r_y, y_{ss} = C x_{ss} = C N_x r_y$$

$$\therefore r_y = C N_x r_y$$

$$C N_x = I$$

$$\begin{bmatrix} (A - I) & B \\ C & 0 \end{bmatrix} \begin{bmatrix} N_x \\ N_u \end{bmatrix} = \begin{bmatrix} 0 \\ I \end{bmatrix}$$

Integral Control for State Space Models

State space control, " $u = -Kx$ ", involves multiplication of state vectors, x , by a matrix of gains, K . Thus, we can capture a "PD" (proportional plus derivative) controller if there are states to represent position and velocity: $u = -K_1 * x - K_2 * dx/dt$

To include an "integral action", for example "PID" control, we need an (internal) state that is the INTEGRAL of another (actual) state of the dynamic system.

Answer: Augment the system with an additional state! Let us call this new state: x_I

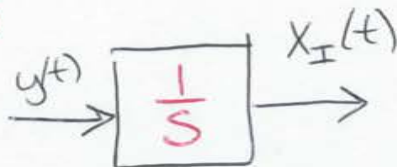
S-domain

Continuous Time: $x_I(t)$

$$x_I(t) = \int_0^t y(t) dt$$

$$x_I = \frac{1}{s} y$$

$$\frac{x_I}{y} = \frac{1}{s}$$



Discrete Time: $x_I(k)$

z-domain

$$x_I(k+1) = x_I(k) + y(k)$$

$$z \cdot x_I = x_I + y$$

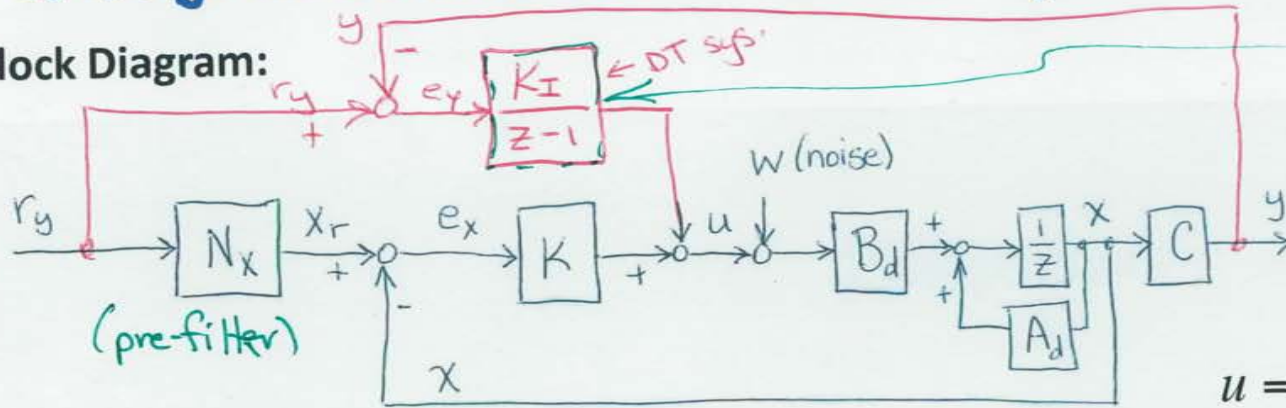
$$(z-1) \cdot x_I = y$$

$$\therefore \frac{x_I}{y} = \frac{1}{z-1}$$



Integral Control for State Space Models

Block Diagram:



$\frac{K_I}{s}$ for a CT sys.

$$u = -[K_I \quad K] \begin{bmatrix} x_I \\ x \end{bmatrix} + K N_x r_y$$

Discrete-time (DT) state space equations:

$$"u = -K_I x_I - Kx" + K N_x r_y$$

Was: $zX = AX + Bu \dots$

Continuous-time (CT) state space equations:

$$z \begin{bmatrix} x_I \\ x \end{bmatrix} = \begin{bmatrix} I & C \\ 0 & A \end{bmatrix} \begin{bmatrix} x_I \\ x \end{bmatrix} + \begin{bmatrix} 0 \\ B \end{bmatrix} u - \begin{bmatrix} I \\ 0 \end{bmatrix} r$$

$y = Cx$

b/c we operate on "e_y"

for CT, this term (these terms) are 0.

$$s \begin{bmatrix} x_I \\ x \end{bmatrix} = \begin{bmatrix} 0 & C \\ 0 & A \end{bmatrix} \begin{bmatrix} x_I \\ x \end{bmatrix} + \begin{bmatrix} 0 \\ B \end{bmatrix} u - \begin{bmatrix} I \\ 0 \end{bmatrix} r$$

$\frac{1}{s}$ vs $\frac{1}{z-1}$

Integral Control for State Space Models

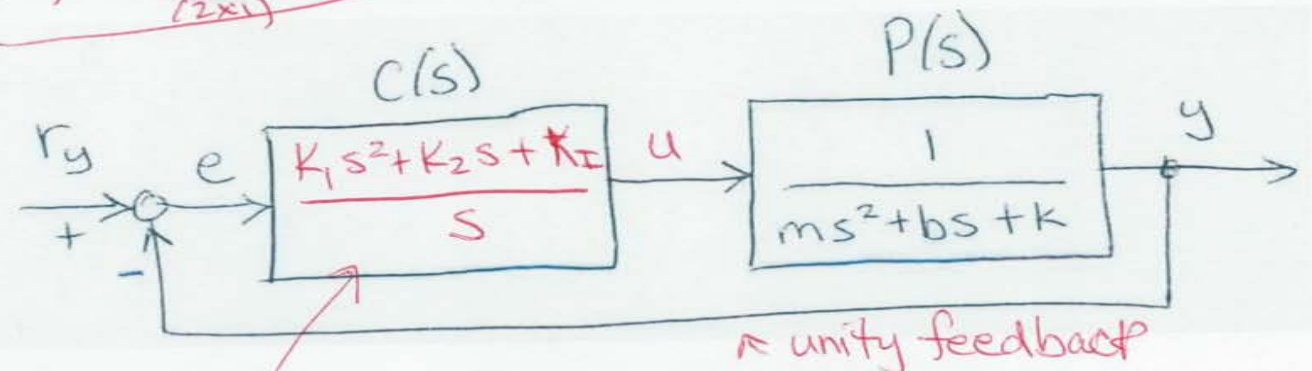
Example: spring-mass-damper system (2nd order). Continuous time.

n states, l outputs: $n+l$ total poles (since integral of each output is a NEW state now).

$$X = \begin{bmatrix} \dot{x} \\ x \end{bmatrix} \quad C = \begin{bmatrix} 0 & 1 \end{bmatrix} \quad N_x = \begin{bmatrix} 0 \\ 1 \end{bmatrix} \quad A = \begin{bmatrix} -b/m & -k/m \\ 1 & 0 \end{bmatrix} \quad B = \begin{bmatrix} 1/m \\ 0 \end{bmatrix}$$

Control canonical form

$$P(s) = \left(\frac{1}{ms^2 + bs + k} \right)$$



Closed-loop system?

$$C(s) = \frac{K_I}{s} + K_1 s + K_2$$

$$= \frac{K_1 s^2 + K_2 s + K_I}{s}$$

$$K_1 s^2 + K_2 s + K_I$$

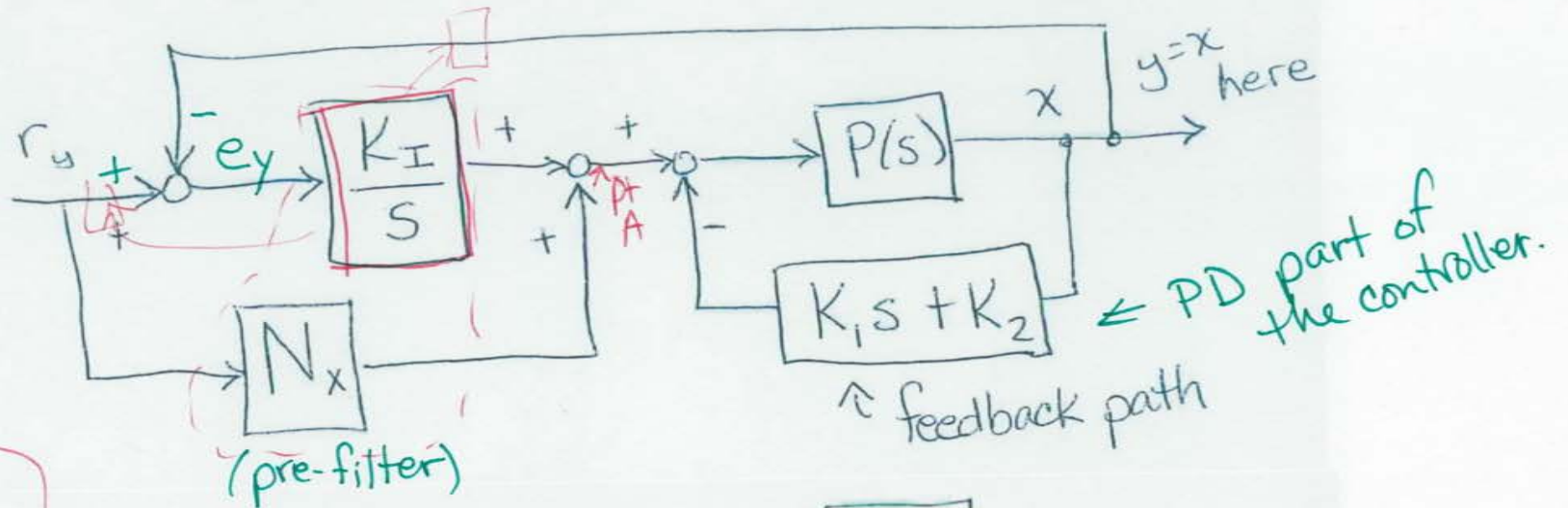
$$ms^3 + (b + K_1)s^2 + (k + K_2)s + K_I$$

@ ss, response to a unit step?

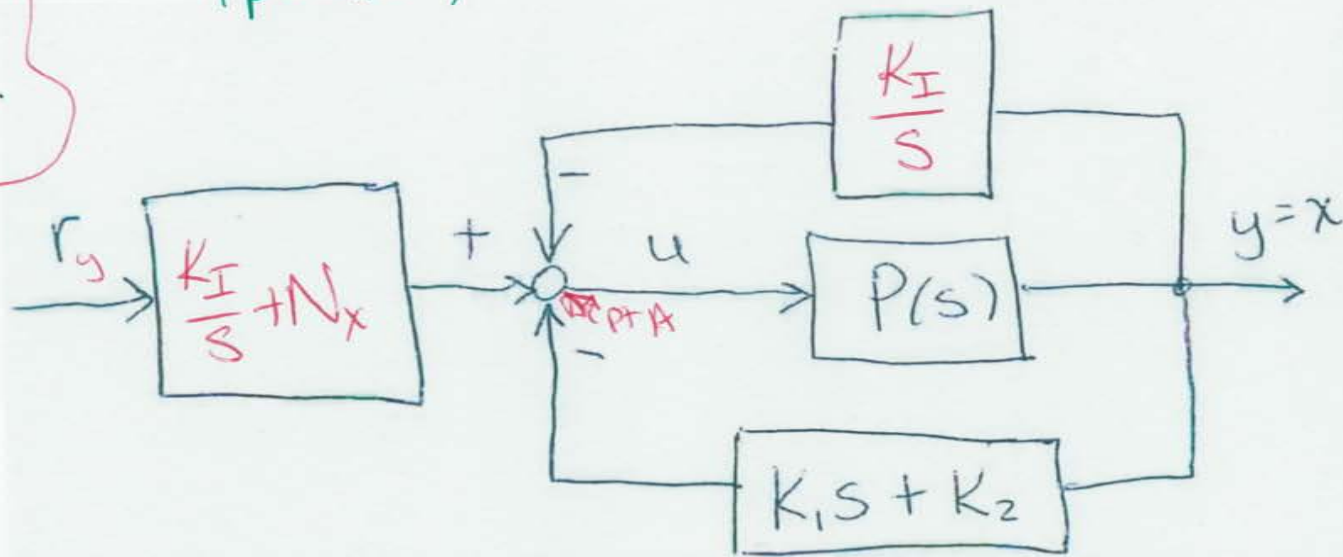
$$\frac{K_I}{K_I} = 1 \quad (\text{no error!})$$

Integral Control for State Space Models

Alternate form, with the "PD" part in the FEEDBACK path, instead of the FORWARD path:

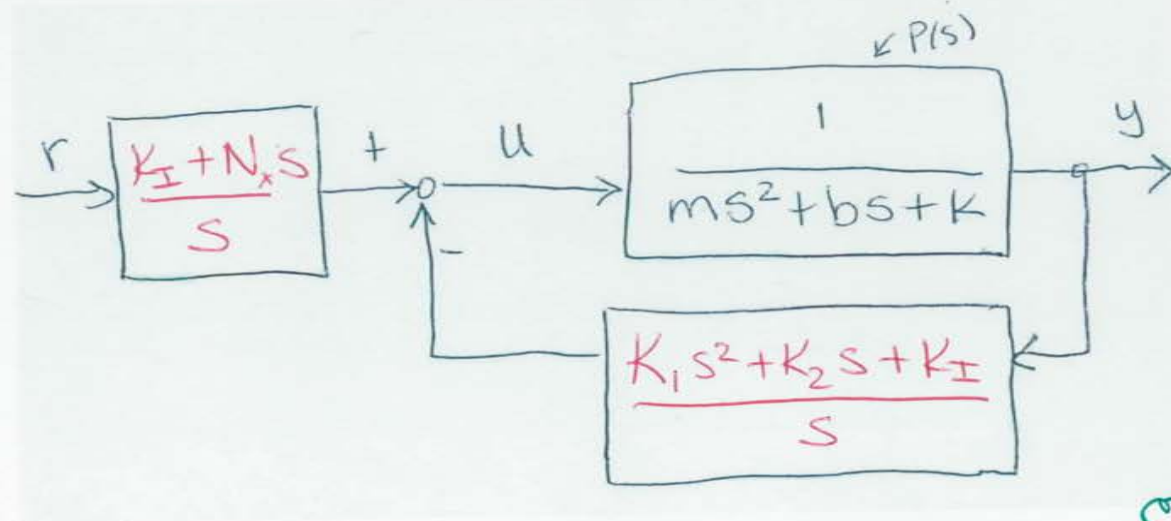


Block
Diagram
Algebra...



Integral Control for State Space Models

Alternate form,
continued...



For DT sys,
zero is

$$z = 1 - \frac{K_I}{N_x}$$

Different
zeros,

Closed-loop system?

zero @

$$s = -\frac{K_I}{N_x}$$

zero
pole

$$(N_x s + K_I)$$

$$ms^3 + (b + K_1)s^2 + (k + K_2)s + K_I$$

same char. eqn.
same poles

Integral Control for State Space Models

Recall... n states, l outputs: $(n+1)$ total poles.

Notes:

1. Putting control dynamics (PD part) in the feedback path can reduce overshoot. (i.e., Avoid some or all of the "zeros".)

2. Can place pole(s) to 'cancel out' the l zeros due to integral control, and then set the remaining n poles (via pole placement) as desired.

$\oint \epsilon \omega_n$, as
desired,
for dominant poles.