

HW#2 Solutions

1. (a) e_{yy} and e_{zz} are identically zero from the definition of “clamped”
 (b) To derive the stress terms, we first write the form of the stiffness matrix from the clues given for a cubic solid:

$$\begin{pmatrix} C_{11} & C_{12} & C_{12} & 0 & 0 & 0 \\ C_{12} & C_{11} & C_{12} & 0 & 0 & 0 \\ C_{12} & C_{12} & C_{11} & 0 & 0 & 0 \\ 0 & 0 & 0 & C_{44} & 0 & 0 \\ 0 & 0 & 0 & 0 & C_{44} & 0 \\ 0 & 0 & 0 & 0 & 0 & C_{44} \end{pmatrix}$$

By inspection, $P_{xx} = C_{11}e_{xx} + C_{12}e_{yy} + C_{12}e_{zz} = C_{11}e_{xx}$, $P_{zz} = C_{12}e_{xx} + C_{12}e_{yy} + C_{11}e_{zz} = C_{12}e_{xx}$.

- (c) Assuming $e_{xx} = 1 \times 10^{-2}$, $C_{11} = 1.66 \times 10^{11} \text{ N/m}^2$, $C_{12} = 0.64 \times 10^{11} \text{ N/m}^2$ we get

$$P_{xx} = 1.6 \times 10^9 \text{ N/m}^2, P_{zz} = 0.64 \times 10^9 \text{ N/m}^2$$

- (d) If replaced by an isotropic homogeneous solid, we still have 12 nonzero stiffness matrix elements. But there are now only two independent ones.
 (e) if clamps are removed, $P_{xx} = Y e_{xx}$ where Y is Young's modulus. $e_{zz} = e_{yy} = \sigma e_{xx} = \sigma P_{xx}/Y$.

2. (a) Given the following form of the compliance matrix,

$$\begin{pmatrix} 0.7664 & -0.2130 & -0.2130 & 0 & 0 & 0 \\ -0.2130 & 0.7664 & -0.2130 & 0 & 0 & 0 \\ -0.2130 & -0.2130 & 0.7664 & 0 & 0 & 0 \\ 0 & 0 & 0 & 1.2563 & 0 & 0 \\ 0 & 0 & 0 & 0 & 1.2563 & 0 \\ 0 & 0 & 0 & 0 & 0 & 1.2563 \end{pmatrix}$$

we expect the inverse (stiffness) matrix to have exactly the same form; i.e., all nonzero elements in the upper-left quadrant; all zeroes in the lower left and upper right quadrants; and nonzero elements only along the diagonal of the lower-right quadrant.

- (b) substitution of this matrix into Matlab and using the inv.m function (matrix inversion) leads to the result.

$$\begin{pmatrix} 1.6600 & 0.6390 & 0.6390 & 0 & 0 & 0 \\ 0.6390 & 1.6600 & 0.6390 & 0 & 0 & 0 \\ 0.6390 & 0.6390 & 1.6600 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0.7960 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0.7960 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0.7960 \end{pmatrix}$$

Assuming the units are 10^{11} N/m^2 , we have $C_{11} = 166 \text{ GPa}$, $C_{12} = 63.9 \text{ GPa}$, and $C_{44} = 79.6 \text{ GPa}$ - the three distinct coefficient for crystalline silicon !

3. (a) $P_{xx} = \text{stress component} = +10 \text{ N/Area} = +10/10^{-4} = +10^5 \text{ N/m}^2$; the positive sign because the force is tensile. By definition $e_{xx} = P_{xx}/Y$ along the x axis. So the strain is given by $e_{xx} = 10^5 / 1.3 \times 10^{11} \text{ N/m}^2 = 7.7 \times 10^{-7}$.
 (b) Both lateral strains are given by $e_{yy} = e_{zz} = \sigma P_{xx}/Y = \sigma e_{xx} = 0.28 \cdot 7.7 \times 10^{-7} = 2.2 \times 10^{-7}$.

4. (a) In the text the general form of the compliance matrix for an isotropic solid is found:

$$\begin{bmatrix} e_{xx} \\ e_{yy} \\ e_{zz} \\ e_{yz} \\ e_{xz} \\ e_{xy} \end{bmatrix} = \begin{bmatrix} 1/Y & -\sigma/Y & -\sigma/Y & 0 & 0 & 0 \\ -\sigma/Y & 1/Y & -\sigma/Y & 0 & 0 & 0 \\ -\sigma/Y & -\sigma/Y & 1/Y & 0 & 0 & 0 \\ 0 & 0 & 0 & 2(1+\sigma)/Y & 0 & 0 \\ 0 & 0 & 0 & 0 & 2(1+\sigma)/Y & 0 \\ 0 & 0 & 0 & 0 & 0 & 2(1+\sigma)/Y \end{bmatrix} \bullet \begin{bmatrix} P_{xx} \\ P_{yy} \\ P_{zz} \\ P_{yz} \\ P_{xz} \\ P_{xy} \end{bmatrix}$$

We expect the stiffness matrix to have the form:

$$\begin{bmatrix} P_1 \\ P_2 \\ P_3 \\ P_4 \\ P_5 \\ P_6 \end{bmatrix} = \begin{bmatrix} C_{11} & C_{12} & C_{12} & 0 & 0 & 0 \\ C_{12} & C_{11} & C_{12} & 0 & 0 & 0 \\ C_{12} & C_{12} & C_{11} & 0 & 0 & 0 \\ 0 & 0 & 0 & C_{44} & 0 & 0 \\ 0 & 0 & 0 & 0 & C_{44} & 0 \\ 0 & 0 & 0 & 0 & 0 & C_{44} \end{bmatrix} \bullet \begin{bmatrix} e_1 \\ e_2 \\ e_3 \\ e_4 \\ e_5 \\ e_6 \end{bmatrix}$$

By definition, the stiffness matrix is the inverse of the compliance matrix. Stated mathematically, this means that if X is the matrix product of the two matrices, then

$$X_{MN} = \sum_K C_{MK} S_{KN} = \delta_{M,N} \text{ where } \delta_{M,N} \text{ is the Kronecker delta function } (= 1 \text{ if } M = N; = 0 \text{ if } M \neq N).$$

It is easy to show $X_{11} = X_{22} = X_{33} = 1 = (C_{11} - 2C_{12}\sigma)/Y$ (*), and $X_{12} = X_{13} = X_{21} = X_{23} = X_{31} = X_{32} = 0 = [C_{12} - \sigma(C_{11} + C_{12})]/Y$ (**). Multiplying (*) by σ and adding to (**) (to eliminate dependence on C_{11}), we get $\sigma Y = -2C_{12}\sigma^2 + C_{12}(1 - \sigma)$, or

$$C_{12} = \sigma Y / (1 - \sigma - 2\sigma^2).$$

Substitution of this back yields $C_{11} = Y + 2\sigma^2 Y / (1 - \sigma - 2\sigma^2)$, or

$$C_{11} = Y(1 - \sigma) / (1 - \sigma - 2\sigma^2)$$

To get C_{22} , we note $X_{44} = X_{55} = X_{66} = 1 = C_{44} 2(1+\sigma)/Y$, or

(b) In the notes an alternative form of the stiffness matrix is derived for an isotropic solid in terms of Lamé coefficients

$$\begin{bmatrix} P_{xx} \\ P_{yy} \\ P_{zz} \\ P_{yz} \\ P_{xz} \\ P_{xy} \end{bmatrix} = \begin{bmatrix} 2\mu_L + \lambda_L & \lambda_L & \lambda_L & 0 & 0 & 0 \\ \lambda_L & 2\mu_L + \lambda_L & \lambda_L & 0 & 0 & 0 \\ \lambda_L & \lambda_L & 2\mu_L + \lambda_L & 0 & 0 & 0 \\ 0 & 0 & 0 & \mu_L & 0 & 0 \\ 0 & 0 & 0 & 0 & \mu_L & 0 \\ 0 & 0 & 0 & 0 & 0 & \mu_L \end{bmatrix} \bullet \begin{bmatrix} e_{xx} \\ e_{yy} \\ e_{zz} \\ e_{yz} \\ e_{xz} \\ e_{xy} \end{bmatrix}$$

These can be related directly to the Young's modulus and Poisson's ratio through the stiffness coefficients. By inspection, we have

$$\lambda_L = C_{12} = \sigma Y / (1 - \sigma - 2\sigma^2)$$

$$\mu_L = Y / [2(1 + \sigma)]$$

As a self consistency check we take the sum $2\mu_L + \lambda_L = Y(1 - \sigma) / (1 - \sigma - 2\sigma^2) = C_{11}$, as expected.

5. (a) For purified aluminum $Y = 73 \text{ GPa}$ and $\sigma = 0.33 \Rightarrow \lambda_L = 53.3 \text{ GPa}$ and $\mu_L = 27.4 \text{ GPa}$.

For amorphous SiO₂, $Y = 85$ GPa, and $\sigma = 0.25 \Rightarrow \lambda_L = 34.0$ GPa and $\mu_L = 34.0$ GPa (equal !)
 (b) From the Lamé coefficients for aluminum, $C_{12} = \lambda_L = 53$ GPa, $C_{11} = 2\mu_L + \lambda_L = 108$ GPa and $C_{44} = \mu_L = 27$ GPa. From the Lamé coefficients for amorphous SiO₂, $C_{12} = 34$ GPa, $C_{11} = 102$ GPa and $C_{44} = 34$ GPa.

6. The generic equation to be solved is

$$\frac{\partial^2 F}{\partial x^2} - \frac{1}{v^2} \frac{\partial^2 F}{\partial t^2} = 0$$

(a). Letting $u = x + vt$, $w = x - vt$, we get

$$\frac{\partial F}{\partial x} = \frac{\partial F}{\partial u} \frac{\partial u}{\partial x} + \frac{\partial F}{\partial w} \frac{\partial w}{\partial x} = \frac{\partial F}{\partial u} + \frac{\partial F}{\partial w}$$

$$\begin{aligned} \frac{\partial^2 F}{\partial x^2} &= \frac{\partial}{\partial x} \left(\frac{\partial F}{\partial u} + \frac{\partial F}{\partial w} \right) = \frac{\partial}{\partial u} \frac{\partial F}{\partial x} + \frac{\partial}{\partial w} \frac{\partial F}{\partial x} \\ &= \frac{\partial^2 F}{\partial u^2} + \frac{\partial^2 F}{\partial u \partial w} + \frac{\partial^2 F}{\partial w \partial u} + \frac{\partial^2 F}{\partial w^2} = \frac{\partial^2 F}{\partial u^2} + 2 \frac{\partial^2 F}{\partial u \partial w} + \frac{\partial^2 F}{\partial w^2} \end{aligned}$$

(b)

$$\frac{\partial^2 F}{\partial t^2} = v \frac{\partial}{\partial u} \frac{\partial F}{\partial t} - v \frac{\partial}{\partial w} \frac{\partial F}{\partial t} = v^2 \frac{\partial^2 F}{\partial u^2} - 2v^2 \frac{\partial^2 F}{\partial u \partial w} + v^2 \frac{\partial^2 F}{\partial w^2}$$

(c) Given the results of (a) and (b), the new wave equation is

$$\frac{\partial^2 F}{\partial u^2} + 2 \frac{\partial^2 F}{\partial u \partial w} + \frac{\partial^2 F}{\partial w^2} - \frac{\partial^2 F}{\partial u^2} + 2 \frac{\partial^2 F}{\partial u \partial w} - \frac{\partial^2 F}{\partial w^2} = 4 \frac{\partial^2 F}{\partial u \partial w} = 0$$

(d) The solution is obtained by integrating with respect to u and w

$$\int 4 \frac{\partial^2 F}{\partial u \partial w} dw = C_1 G(u) + C_2, \text{ where } C_1 \text{ and } C_2 \text{ are constants, } G \text{ an arbitrary function.}$$

So we can write $\int 4 \frac{\partial^2 F}{\partial u \partial w} dw du = C_1 H(u) + C_4 + C_3 I(w)$, where C_2, C_3, C_4 are constants, and $H(u)$ and $I(w)$ are arbitrary functions. Hence, the general solution is:

$$F = C_2 H(x+vt) + C_3 I(x-vt) + C_4.$$

7. (a). along the $\mathbf{x}$, $\mathbf{y}$, or $\mathbf{z}$ directions in an isotropic solid, we can use the same formulation as for a cubic solid and solve the component wave equations from the text. With a longitudinal wave $\Delta r_x = \Delta r_0 \exp[j(kx - \omega t)]$ along the x axis, for example. The solution leads to the dispersion relation $\omega = k c = k (C_{11}/\rho)^{1/2} = k [(2\mu_L + \lambda_L)/\rho]^{1/2}$ where c is the speed of longitudinal sound in terms of the stiffness and Lamé coefficients.

(b). For the shear wave, we solve for $r_y = \Delta r_0 \exp[j(kx - \omega t)]$ or

$$r_z = \Delta r_0 \exp[j(kx - \omega t)] \text{ and find } \omega = k c = k (C_{44}/\rho)^{1/2}.$$

(c) Along the $\mathbf{x} + \mathbf{y} + \mathbf{z}$ axis (cube diagonal in Cartesian coordinates), we let

$u = \Delta r_x = v = \Delta r_y = z = \Delta r_z$ then the elastic wave equation becomes

$$\rho \frac{\partial^2 u}{\partial t^2} = C_{11} \frac{\partial^2 u}{\partial x^2} + C_{44} \left(\frac{\partial^2 u}{\partial y^2} + \frac{\partial^2 u}{\partial z^2} \right) + (C_{12} + C_{44}) \left(\frac{\partial^2 u}{\partial x \partial y} + \frac{\partial^2 u}{\partial x \partial z} \right)$$

(only dependent variable is u !). We seek the solution $u = u_o e^{[j(k(x+y+z)/\sqrt{3} - \omega t)]}$ and

substitute to get

$$-\rho \omega^2 u_o = -C_{11} \frac{k^2}{3} u_o - C_{44} \left(\frac{k^2}{3} + \frac{k^2}{3} \right) u_o - (C_{12} + C_{44}) \left(\frac{k^2}{3} + \frac{k^2}{3} \right) u_o$$

$$\rho \omega^2 = C_{11} \frac{k^2}{3} + 4C_{44} \frac{k^2}{3} + 2C_{12} \frac{k^2}{3}$$

$$\text{or } \frac{\omega}{k} = v_s = \left[\frac{1}{\rho} \left(\frac{C_{11}}{3} + \frac{2C_{12}}{3} + \frac{4C_{44}}{3} \right) \right]^{1/2} = \frac{\omega}{k} = v_s = \left[\frac{1}{\rho} \left(\frac{C_{11}}{3} + \frac{2C_{12}}{3} + \frac{4C_{44}}{3} \right) \right]^{1/2}$$

(d). To get the velocities we first need the densities: a little searching reveals $\rho = 2700 \text{ KG/m}^3$ (Al) and $\rho = 2600 \text{ KG/m}^3$ (SiO_2), so that for compressional wave along $\mathbf{x}$, $\mathbf{y}$, or $\mathbf{z}$:

$$c = k(C_{11}/\rho)^{1/2} = [(2\mu_L + \lambda_L)/\rho]^{1/2} = 6325 \text{ m/s for aluminum and } 6263 \text{ m/s for SiO}_2$$

For the shear wave along $\mathbf{x}$, $\mathbf{y}$, or $\mathbf{z}$: we get $c = k(C_{44}/\rho)^{1/2} = [\mu_L/\rho]^{1/2} = 3162 \text{ m/s for Al and}$

$3616 \text{ m/s for SiO}_2$. For the compressional wave along $\mathbf{x} + \mathbf{y} + \mathbf{z}$ direction, we get

$$\frac{\omega}{k} = v_s = \left[\frac{1}{\rho} \left(\frac{C_{11}}{3} + \frac{2C_{12}}{3} + \frac{4C_{44}}{3} \right) \right]^{1/2} = \left[\frac{1}{\rho} \left(\frac{2\mu_L + \lambda_L}{3} + \frac{2\lambda_L}{3} + \frac{4\mu_L}{3} \right) \right]^{1/2} = \left[\frac{2\mu_L + \lambda_L}{\rho} \right]^{1/2}$$

So the longitudinal wave is the same along the $\mathbf{x} + \mathbf{y} + \mathbf{z}$ as along $\mathbf{x}$, $\mathbf{y}$, or $\mathbf{z}$. This can be generalized to a compressional (or shear) wave velocity independent of orientation, just what one expects in an “isotropic” solid !

8. Clamped AlN film is “clamped” in the x - y plane and subjected to a uniform external electric field of $E_0 = 10 \text{ KV/cm}$ ($=10^6 \text{ V/m}$) along the z axis.

(a). By definition, clamped in x and y means that there can be no strain in these axes and the only nonzero strain term will be η_3 . So the connection equation will be the inverse piezoelectric

relation $P_J = C_{IJ} \eta_J - e_{ij} E_j$. To use this, we need the 6x3 form of the piezoelectric stress which can be derived from the more commonly used 3x6 form as follows:

$$\begin{pmatrix} 0 & 0 & 0 & 0 & e_{15} & 0 \\ 0 & 0 & 0 & e_{15} & 0 & 0 \\ e_{31} & e_{31} & e_{33} & 0 & 0 & 0 \end{pmatrix} \Rightarrow \begin{bmatrix} 0 & 0 & e_{31} \\ 0 & 0 & e_{31} \\ 0 & 0 & e_{33} \\ 0 & e_{15} & 0 \\ e_{15} & 0 & 0 \\ 0 & 0 & 0 \end{bmatrix}$$

We also need the stiffness coefficients and values of the piezoelectric stress coefficients, which are both tabulated here:

AlN Stiffness Coefficients ($\times 10^{10}$ Newton/m ²)						
C11	C12	C13	C14	C33	C44	C66
41	14	10		39	12	13.5
Piezoelectric Constants (Coulomb/m ²)						
Material	e11	e14	e15	e2 2	e31	e33
AlN			-0.48		-0.58	1.55

The stiffness coefficients go into the general stiffness matrix for solids having hexagonal symmetry:

$$\begin{bmatrix} C_{11} & C_{12} & C_{13} & 0 & 0 & 0 \\ 0 & C_{11} & C_{13} & 0 & 0 & 0 \\ 0 & 0 & C_{33} & 0 & 0 & 0 \\ 0 & 0 & 0 & C_{44} & 0 & 0 \\ 0 & 0 & 0 & 0 & C_{44} & 0 \\ 0 & 0 & 0 & 0 & 0 & C_{66} \end{bmatrix}$$

where $C_{66} = (1/2)(C_{11} - C_{12})$. Now using matrix multiplication, we can write

$$P_1 = C_{13} \eta_3 - e_{31} E_0$$

$$P_2 = C_{13} \eta_3 - e_{31} E_0$$

$$P_3 = C_{33} \eta_3 - e_{33} E_0$$

Where the other possible P components: P_4 , P_5 , and P_6 are necessarily zero because we assumed up front that the only nonzero strain component was η_3 . Substitution of the appropriate values leads to

$$P_3 = 39 \times 10^{10} \eta_3 - 1.55 \times 10^6 \quad [\text{N/m}^2]$$

(b) To get the surface charge density in the two interface planes, we start with the direct piezoelectric relation: $D_i = \epsilon_{ij} E_j + e_{ij} \eta_j$. When expressed in terms of the known E_z and η_3 , and using the above 3x6 form of the piezoelectric stiffness matrix, we get $D_z = \epsilon_r \epsilon_0 E_0 + e_{33} \eta_3$. Then from electrostatics, we know $P_z = D_z - \epsilon_0 E_0 = (\epsilon_r - 1) \epsilon_0 E_0 + e_{33} \eta_3 = 6.6 \times 10^{-5} + 1.55 \eta_3$ [Cb/m²]. Until we know the strain, no further evaluation can be done.

(c) If the AlN film is now used as a MEMS cantilever by unclamping it, by definition all the stress coefficients will be zero. So we can write from the piezoelectric inverse relation,

$P_j = 0 = C_{ij} \eta_j - e_{ij} E_j$, which in full matrix form becomes

$$\begin{bmatrix} C_{11} & C_{12} & C_{13} & 0 & 0 & 0 \\ C_{12} & C_{11} & C_{13} & 0 & 0 & 0 \\ C_{13} & C_{13} & C_{33} & 0 & 0 & 0 \\ 0 & 0 & 0 & C_{44} & 0 & 0 \\ 0 & 0 & 0 & 0 & C_{44} & 0 \\ 0 & 0 & 0 & 0 & 0 & C_{66} \end{bmatrix} \begin{bmatrix} \eta_1 \\ \eta_2 \\ \eta_3 \\ \eta_4 \\ \eta_5 \\ \eta_6 \end{bmatrix} = \begin{bmatrix} 0 & 0 & e_{31} \\ 0 & 0 & e_{31} \\ 0 & 0 & e_{33} \\ 0 & e_{15} & 0 \\ e_{15} & 0 & 0 \\ 0 & 0 & 0 \end{bmatrix} \begin{bmatrix} 0 \\ 0 \\ E_z \end{bmatrix}$$

And when written out using the stress coefficients and $E_0 = 10^6$ V/m becomes

$$C_{11} \eta_1 + C_{12} \eta_2 + C_{13} \eta_3 = e_{31} E_z = -0.58 \times 10^6$$

$$C_{12} \eta_1 + C_{11} \eta_2 + C_{13} \eta_3 = e_{31} E_z = -0.58 \times 10^6$$

$$C_{13} \eta_1 + C_{13} \eta_2 + C_{33} \eta_3 = e_{33} E_z = 1.55 \times 10^6$$

This has a simple 3x3 numerical matrix form: $10^{10} \cdot \begin{bmatrix} 41 & 14 & 10 \\ 14 & 41 & 10 \\ 10 & 10 & 39 \end{bmatrix} \begin{bmatrix} \eta_1 \\ \eta_2 \\ \eta_3 \end{bmatrix} = \begin{bmatrix} -0.58 \times 10^6 \\ -0.58 \times 10^6 \\ 1.55 \times 10^6 \end{bmatrix}$

(d) One can use MATLAB to invert the 3x3 numerical matrix to get

$$\begin{bmatrix} \eta_1 \\ \eta_2 \\ \eta_3 \end{bmatrix} = 10^{-10} \cdot \begin{bmatrix} 0.0285 & -0.0085 & -0.0051 \\ -0.0085 & 0.0285 & -0.0051 \\ -0.0051 & -0.0051 & 0.0283 \end{bmatrix} \begin{bmatrix} -0.58 \times 10^6 \\ -0.58 \times 10^6 \\ 1.55 \times 10^6 \end{bmatrix}$$

or $\eta_1 = -1.96 \times 10^{-6}$, $\eta_2 = -1.96 \times 10^{-6}$, and $\eta_3 = 4.98 \times 10^{-6}$